Volume: 4 Issue: 4 35 - 38

Investigation of Muffler for Noise Reduction

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Abstract- Internal ignition motor is a noteworthy wellspring of commotion contamination. Fumes commotion lessening from motors is a vital issue. Suppressors are utilized to diminish this fumes commotion. Configuration of the suppressor ought to give the best commotion diminishment and offer ideal backpressure for the motor. In this paper the Experimental and reproduction results for altered suppressor are exhibited. The standard suppressor with various change are displayed in Autodesk Inventor programming and tried for acoustical execution in COMSOL a reproducing programming. Clamor estimation for various suppressor alterations are completed tentatively utilizing Fast Fourier Transform. Relatively changed intelligent suppressor with astounds demonstrates decrease in clamor level and least fuel utilization.

Keywords—Noise, Muffler, FFT Analyser, COMSOL

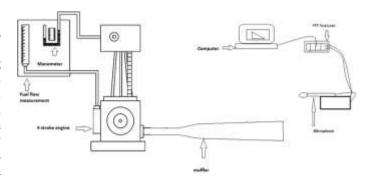
I. Introduction

The noise created by internal combustion engine has been a constant source of problem to the atmosphere. So the problems of reducing engine noise consist of mainly the attenuating exhaust noise. Exhaust systems are developed to attenuate noise levels to meet prescribed levels and sound quality, and emissions based on environment norms. Mufflers are an important part of engine system and are commonly used in exhaust system to minimize sound transmissions caused by exhaust gases. A resistance against the flow of exhaust gases stems the flow is called as back pressure and it causes an extra pressure inside the engine. Because of the back pressure, volumetric efficiency decreases and specific fuel consumption increases. Therefore, there must be specific limitations for the back pressure. Noise pollution created by engines becomes a vital concern when used in residential areas or areas where noise creates hazard. Generally, noise level of more than 80 dB is not comfortable for human being. Design of mufflers is a complex function that affects emission and fuel efficiency of engine. Therefore, muffler design has become important for noise reduction, making this an important area of research and development.

Particularly focusing on the engine under study, this paper tackles the problem of the inconvenience caused due to the noise produced by the engine by determining a suitable muffler design for it by actual modification and by doing acoustical analysis of 3D model of muffler in COMSOL software. Comparative study of the muffler modifications helps to determine the best design type for noise reduction, taking into consideration the effect of the design on fuel economy.

II. Methodology

Microphone (Bruel & Kjaer make) & FFT SN-28995 is used for measurement of Sound pressure level. The experiment has been carried out in an open environment. The Figure 2.1 shows experimental setup for noise measurement.



ISSN: 2321-8169

Fig.2.1 Experimental setup

Hardware components in the system are noise source, sensor and FFT system. The engine generates the noise level which is to be measured and the sensor and the FFT system are used to measure the noise level and to convert it into a computer understandable format.

The single cylinder 4-stroke petrol engine used for the experiment has the following specifications:

- Make Bajaj Auto Ltd.
- Engine Capacity 5 HP at 1500 rpm
- Capacity 100cc
- Rated speed 1500 rpm
- Rated BHP 6 K.W.

Measurements are carried out when no other major noise sources are in action. The background noise was neglected and is assumed to be in a constant range for all readings.

Noise level of the engine is first measured without attaching a muffler, then with a standard muffler and afterwards with different modifications. The standard muffler used for experiment is of the same make of Bajaj 4S Champion. Measurements are taken at 3 speeds of 2500 rpm, 3500 rpm and 4500 rpm. The time taken for consumption of 10 cm level of fuel is recorded for all modification.

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III. Modifications in muffler design

The standard muffler used for experimental analysis is shown in Fig.3.1 which is purely reflective type with no absorptive material. Three different modifications are done as below.

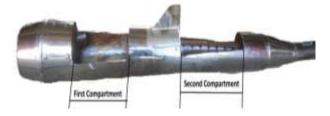


Fig.3.1 Standard muffler

A. Absorptive type muffler

Glass wool packing is used due to its good noise damping properties and low cost. Absorptive mufflers are those mufflers that attenuate sound by using absorbing materials. They dissipate the acoustic energy into heat energy with the use of porous materials like mineral fiber, fiber glass etc. The absorptive muffler is shown in Fig. 3.2



Fig.3.2 Glass wool packing

B. Reflective type muffler (with baffles)

In the first chamber, two baffle plates were placed. The baffle plates used were 80 mm in diameter and 1 mm thick and 4 holes of diameter 12 mm. Placement of baffles is determined by considering the fundamental frequency of emitted noise, so as to cancel the sound waves by reflections. The plates are placed so as to make the three partitions in the chamber of equal length and then different modifications were done by keeping distance between two plates constant and by changing the distance between the first compartment fixed wall and first baffle plate (X). Reflective type modification of muffler with baffle plates is shown in Fig. 3.3.

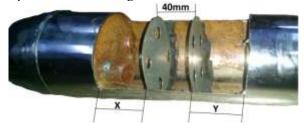


Fig.3.3 Reflective muffler with baffles

C. Hybrid muffler (with baffles)

A combination of the above two modifications, with the absorptive part in the second chamber and the reflective part in the first chamber made up the hybrid muffler.

IV. Experimental and simulation Results

Acoustic simulation of all muffler modifications was done in COMSOL software. The experiments were conducted for all modifications and without and with standard muffler, reading were noted at three different speed for the SPL and the time taken for 10cm level of fuel consumption. Fig. 4.1 shows the comparative time taken for 10cm level of fuel consumption for different modifications and it is listed in Table 1.

Table 1 Time (sec) taken for 10cm level of fuel consumption

| Different Conditions | Time (sec) taken for 10cm level of fuel consumption | | |
|-------------------------|---|-------------|-------------|
| Different Conditions | 2500 rpm | 3500 rpm | 4500 rpm |
| Without Engine Noise | - | - | - |
| Without muffler | 73 | 70 | 65 |
| With standard muffler | 71 | 67 | 63 |
| Absorptive modification | 62 | 58 | 55 |
| Reflective modification | 67 | 60 | 62 |
| Hybrid modification | 64 | 62 | 59 |

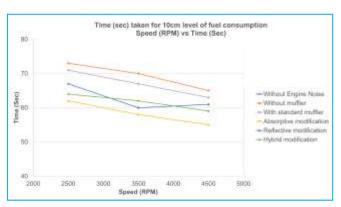


Fig. 4.1 Time (sec) taken for 10cm level of fuel consumption

Fig.4.2 shows comparative sound pressure level for different modifications and Table 2 shows maximum SPL in dB obtained at 600Hz frequency from experimental and simulation results.

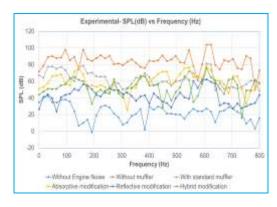


Fig. 4.2 Experimental-SPL(dB) vs Frequency (Hz)

Table 2 SPL (dB) at 600 Hz frequency

| Different Conditions | SPL(dB) | | |
|-------------------------|--------------|------------|--|
| Different Conditions | Experimental | Simulation | |
| Without Engine Noise | 27.09 | - | |
| Without muffler | 103.76 | - | |
| With standard muffler | 81.37 | 78.14 | |
| Absorptive modification | 77.99 | 72.70 | |
| Reflective modification | 62.28 | 62.39 | |
| Hybrid modification | 75.32 | 69.36 | |

Fig. 4.3 and Table 3 shows SPL readings for different modifications of reflective muffler with baffles. It is observed that as the distance X increases the SPL reduces.

Table 3 SPL (dB) for Distance X of reflective modification.

| Baffle distance X (mm) | SPL (dB) Experimental | SPL (dB) Simulation |
|------------------------|--------------------------|------------------------|
| 35 | 73.24 | 75.62 |
| 50 | 68.77 | 72.24 |
| 65 | 62.28 | 62.39 |
| | | |

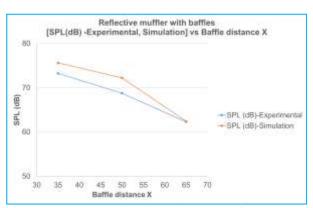


Fig. 4.4 shows acoustic simulation results for standard muffler

Fig. 4.3 SPL (dB) vs Distance X for Reflective modification

and Fig. 4.5 shows acoustic simulation results for reflective muffler.

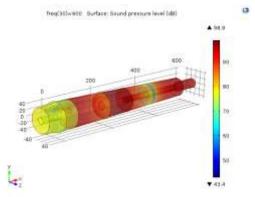


Fig. 4.4 SPL (dB) for standard muffler at 600Hz Frequency in COMSOL (dB) for standard muffler at 600Hz Frequency in COMSOL L

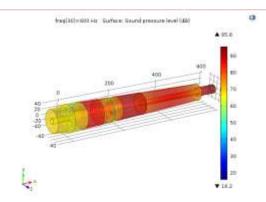


Fig. 4.5 SPL (dB) for reflective muffler at 600Hz Frequency in COMSOL

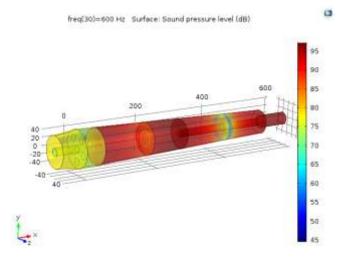


Fig. 4.6 SPL (dB) for absorptive muffler at 600Hz Frequency in COMSOL

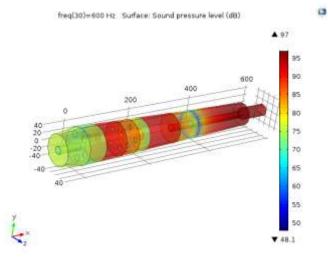


Fig. 4.7 SPL (dB) for hybrid muffler at 600Hz Frequency in COMSOL

V. Conclusion

After experimental investigation and simulation for the Bajaj 4S champion muffler with different modifications like reflective, absorptive and hybrid type for noise reduction it is concluded that the values of noise level and fuel consumption for hybrid type modification lies between reflective and absorptive type muffler. By placing the baffles at optimum

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distance from fixed wall for the reflective type muffler the minimum noise level can be achieved. The reflective muffler with baffles is the better design amongst the four designs studied as it achieves a balance between sound pressure level and fuel consumption.

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